## LINEAR MODELS



#### CONTROL ENGINEERING WITH PYTHON

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### **SYMBOLS**

2	Code		Worked Example
	Graph	**	Exercise
	Definition	AD THE RESERVE OF THE PERSON O	Numerical Method
	Theorem	0000 00 000 0000 000000 0000 000000 000000	Analytical Method
	Remark		Theory
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## **IMPORTS**

```
from numpy import *
from numpy.linalg import *
from scipy.linalg import *
from matplotlib.pyplot import *
from mpl_toolkits.mplot3d import *
from scipy.integrate import solve_ivp
```

## **STREAMPLOT HELPER**

```
def Q(f, xs, ys):
    X, Y = meshgrid(xs, ys)
    v = vectorize
    fx = v(lambda x, y: f([x, y])[0])
    fy = v(lambda x, y: f([x, y])[1])
    return X, Y, fx(X, Y), fy(X, Y)
```

# **PREAMBLE**



## NON-AUTONOMOUS SYSTEMS

#### Their structure is

$$\dot{x} = f(x, u)$$

where  $x \in \mathbb{R}^n$  and  $u \in \mathbb{R}^m$ , that is

$$f:\mathbb{R}^n imes\mathbb{R}^m o\mathbb{R}^n.$$



The vector-valued u is the system input.

This quantity may depend on the time t

$$u:t\in\mathbb{R}\mapsto u(t)\in\mathbb{R}^m,$$

(actually it may also depend on some state, but we will adress this later).



#### A solution of

$$\dot{x}=f(x,u),\;x(t_0)=x_0$$

is merely a solution of

$$\dot x=h(t,x),\; x(t_0)=x_0,$$

where

$$h(t,x) := f(x,u(t)).$$



We may complement the system dynamics with an equation

$$y=g(x,u)\in\mathbb{R}^p$$

The vector y refers to the **systems output**, usually the quantities that we can effectively measure in a system (the state x itself may be unknown).

# **LINEAR SYSTEMS**

#### STANDARD FORM

Input  $u \in \mathbb{R}^m$ , state  $x \in \mathbb{R}^n$ , output  $y \in \mathbb{R}^p$ .

$$\dot{x} = Ax + Bu$$

$$y = Cx + Du$$

### MATRIX SHAPE

$$A \in \mathbb{R}^{n imes n}$$
,  $B \in \mathbb{R}^{n imes m}$ ,  $C \in \mathbb{R}^{p imes n}$ ,  $D \in \mathbb{R}^{p imes m}$ .

$$egin{bmatrix} A & B \ \hline C & D \end{bmatrix}$$



### **WELL-POSEDNESS**

When u=0,

$$\dot{x} = Ax =: f(x) \Rightarrow \frac{\partial f}{\partial x}(x) = A$$

The vector field f is continuously differentiable

 $\Rightarrow$  The system is well-posed.



When u=0, since

$$\dot{x} = Ax =: f(x)$$

$$f(0) = A0 = 0$$

 $\Rightarrow$  the origin x=0 is always an equilibrium.

(the only one in the state space if A is invertible).

## WHY "LINEAR"?

#### Assume that:

$$\dot{x}_1 = Ax_1 + Bu_1, x_1(0) = x_{10},$$

$$\dot{x}_2 = Ax_2 + Bu_2, x_2(0) = x_{20},$$

Set

$$ullet u_3 = \lambda u_1 + \mu u_2$$
 and

• 
$$x_{30} = \lambda x_{10} + \mu x_{20}$$
.

for some  $\lambda$  and  $\mu$ .

Then, if

$$x_3 = \lambda x_1 + \mu x_2,$$

we have

$$\dot{x}_3 = Ax_3 + Bu_3, \ x_3(0) = x_{30}.$$



### **DYNAMICS DECOMPOSITION**

The solution of

$$\dot{x} = Ax + Bu, \ x(0) = x_0$$

is the sum  $x(t) = x_1(t) + x_2(t)$  where

- $x_1(t)$  is the solution to the internal dynamics and
- $x_2(t)$  is the solution to the external dynamics.



• The internal dynamics is controlled by the initial value  $x_0$  only (there is no input, u=0).

$$\dot{x}_1 = Ax_1, \ x_1(0) = x_0,$$

• The external dynamics is controlled by the input u(t) only (the system is initially at rest,  $x_0=0$ ).

$$\dot{x}_2 = Ax_2 + Bu, \; x_2(0) = 0.$$



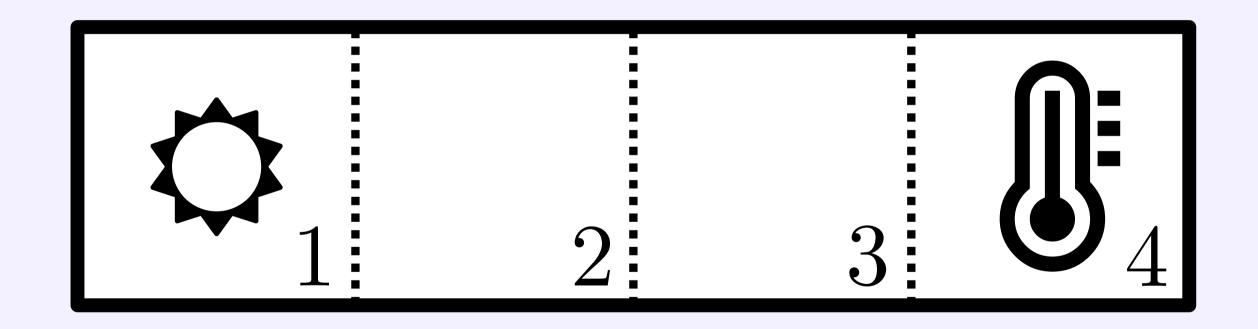
These systems are actually linear and time-invariant (hence LTI) systems. Time-invariant means that when x(t) is a solution of

$$\dot{x}=Ax+Bu,\ x(0)=x_0,$$

then  $x(t-t_0)$  is a solution of

$$\dot{x} = Ax + Bu(t - t_0), \ x(t_0) = x_0.$$

# HEAT EQUATION



### SIMPLIFIED MODEL

- Four cells numbered 1 to 4 are arranged in a row.
- The first cell has a heat source, the last one a temperature sensor.
- The heat sink/source is increasing the temperature of its cell of  $\boldsymbol{u}$  degrees by second.
- If the temperature of a cell is T and the one of a neighbor is  $T_n$ , T increases of  $T_n-T$  by second.

#### Given the geometric layout:

$$\bullet \ dT_1/dt = u + (T_2 - T_1)$$

$$\bullet \ dT_2/dt = (T_1-T_2)+(T_3-T_2)$$

• 
$$dT_3/dt = (T_2 - T_3) + (T_4 - T_3)$$

$$\bullet \ dT_4/dt = (T_3 - T_4)$$

$$ullet y=T_4$$

Set 
$$x = (T_1, T_2, T_3, T_4)$$
.

The model is linear and its standard matrices are:

$$A = egin{bmatrix} -1 & 1 & 0 & 0 \ 1 & -2 & 1 & 0 \ 0 & 1 & -2 & 1 \ 0 & 0 & 1 & -1 \end{bmatrix}$$

$$B = egin{bmatrix} 1 \ 0 \ 0 \ 0 \end{bmatrix}, \ C = [0 \quad 0 \quad 0 \quad 1], \ D = [0]$$

# **LINEARIZATION**

### NONLINEAR TO LINEAR

Consider the nonlinear system

$$\dot{x} = f(x, u)$$

$$y = g(x, u)$$

Assume that  $x_e$  is an equilibrium when  $u=u_e$  (cst):

$$f(x_e, u_e) = 0$$

and let

$$y_e := g(x_e, u_e).$$

#### Define the error variables

$$ullet$$
  $\Delta x := x - x_e,$ 

$$ullet$$
  $\Delta u := u - u_e$  and

• 
$$\Delta y := y - y_e$$
.

As long as the error variables stay small

$$f(x,u)\simeq \overbrace{f(x_e,u_e)}^0 + rac{\partial f}{\partial x}(x_e,u_e)\Delta x + rac{\partial f}{\partial u}(x_e,u_e)\Delta u$$

$$g(x,u)\simeq \overbrace{g(x_e,u_e)}^{y_e} + rac{\partial g}{\partial x}(x_e,u_e)\Delta x + rac{\partial g}{\partial u}(x_e,u_e)\Delta u$$

Hence, the error variables satisfy approximately

$$d(\Delta x)/dt = A\Delta x + B\Delta u$$
  
 $\Delta y = C\Delta x + D\Delta u$ 

with

$$egin{bmatrix} A & B \ \hline C & D \end{bmatrix} = egin{bmatrix} rac{\partial f}{\partial x} & rac{\partial f}{\partial u} \ \hline rac{\partial g}{\partial x} & rac{\partial g}{\partial u} \end{bmatrix} (x_e, u_e)$$



#### The system

$$egin{array}{lll} \dot{x} &=& -2x+y^3 \ \dot{y} &=& -2y+x^3 \end{array}$$

has an equilibrium at (0,0).

The corresponding error variables satisfy  $\Delta x = x$  and  $\Delta y = y$ , thus

$$rac{d\Delta x}{dt} = \dot{x} = -2x + y^3 = -2\Delta x + (\Delta y)^3 pprox -2\Delta x$$

$$rac{d\Delta y}{dt} = \dot{y} = -2y + x^3 = -2\Delta y + (\Delta x)^3 pprox -2\Delta y$$

$$\dot{x} = -2x + y^3$$
 $\dot{y} = -2y + x^3$ 

$$\rightarrow$$

$$egin{array}{lll} \dot{x} &pprox & -2x \ \dot{y} &pprox & -2y \end{array}$$

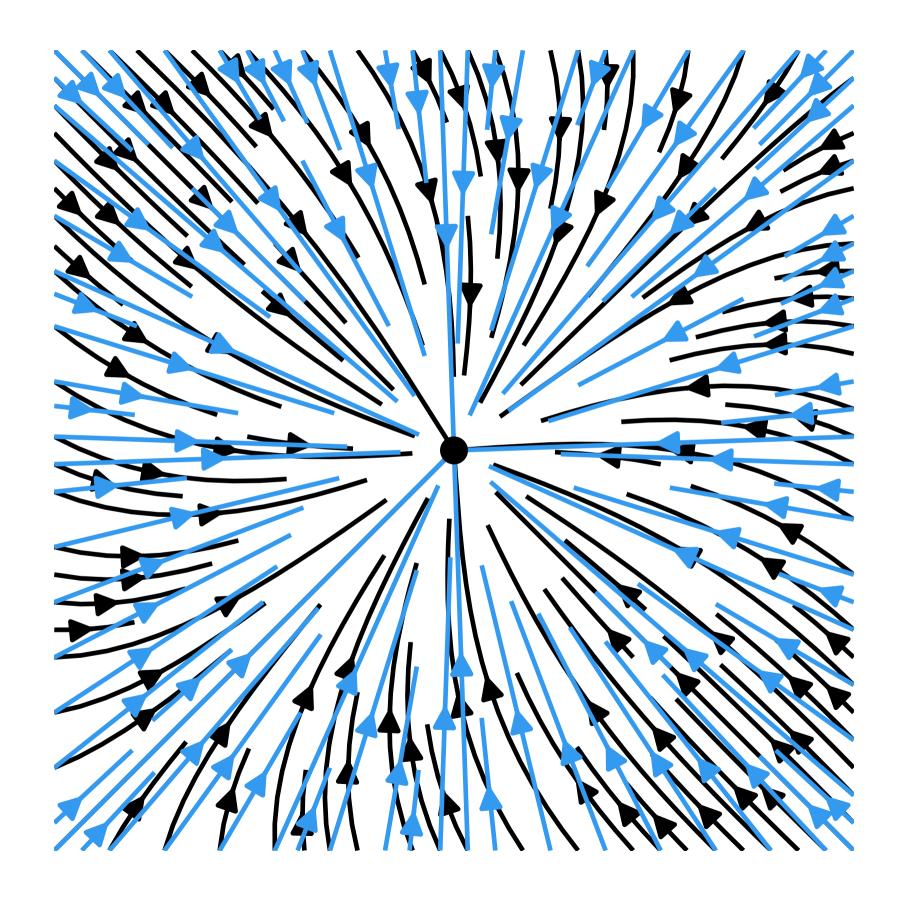
### **\*\* VECTOR FIELDS**

```
def f(xy):
    x, y = xy
    dx = -2*x + y**3
    dy = -2*y + x**3
    return array([dx, dy])
```

```
def fl(xy):
    x, y = xy
    dx = -2*x
    dy = -2*y
    return array([dx, dy])
```

### **STREAM PLOT**

```
figure()
x = y = linspace(-1.0, 1.0, 1000)
streamplot(*Q(f, x, y), color="k")
blue_5 = "#339af0"
streamplot(*Q(f1, x, y), color=blue_5)
plot([0], [0], "k.", ms=10.0)
axis("square")
axis("off")
```





#### Consider

$$\dot{x}=-x^2+u,\;y=xu$$

If we set  $u_e=1$ , the system has an equilibrium at  $x_e=1$  (and also  $x_e=-1$  but we focus on the former) and the corresponding y is  $y_e=x_eu_e=1$ .

Around this configuration  $(x_e,u_e)=(1,1)$ , we have

$$rac{\partial (-x^2+u)}{\partial x}=-2x_e=-2,\;rac{\partial (-x^2+u)}{\partial u}=1,$$

and

$$rac{\partial xu}{\partial x}=u_e=1, \; rac{\partial xu}{\partial u}=x_e=1.$$

Thus, the approximate, linearized dynamics around this equilibrium is

$$d(x-1)/dt = -2(x-1) + (u-1)$$
  
 $y-1 = (x-1) + (u-1)$ 



The equilibrium 0 is locally asymptotically stable for

$$rac{d\Delta x}{dt} = A\Delta x$$

where  $A=\partial f(x_e,u_e)/\partial x$  .

 $\Longrightarrow$ 

The equilibrium  $x_e$  is locally asymptotically stable for

$$\dot{x}=f(x,u_e).$$



- The converse is not true : the nonlinear system may be asymptotically stable but not its linearized approximation (e.g. consider  $\dot{x}=-x^3$ ).
- If we replace local asymptotic stability with local exponential stability, the requirement that locally

$$\|x(t) - x_e\| \leq A e^{-\sigma t} \|x(0) - x_e\|$$

for some A>0 and  $\sigma>0$ , then it works.



A pendulum submitted to a torque c is governed by

$$m\ell^2\ddot{ heta} + b\dot{ heta} + mg\ell\sin{ heta} = c.$$

We assume that only the angle  $\theta$  is measured.

Let  $x=(\theta,\dot{\theta})$ , u=c and  $y=\theta$ .

What are the function f and g that determine the nonlinear dynamics of the pendulum?

Show that for any angle  $heta_e$  there is a constant value  $c_e$  of the torque such that  $x_e=( heta_e,0)$  is an equilibrium.

Compute the linearized dynamics of the pendulum around this equilibrium and put it in the standard form (compute A, B, C and D).

# **PENDULUM**

The 2nd-order differential equation

$$m\ell^2\ddot{ heta} + b\dot{ heta} + mg\ell\sin{ heta} = c.$$

is equivalent to the first-order differential equation

$$rac{d}{dt} egin{bmatrix} heta \ \omega \end{bmatrix} = egin{bmatrix} \omega \ -(b/m\ell^2)\omega - (g/\ell)\sin heta + c/m\ell^2 \end{bmatrix}$$

Hence, with 
$$x=(\theta,\dot{\theta})$$
,  $u=c$  and  $y=\theta$ , we have

$$egin{array}{lll} \dot{x} &=& f(x,u) \ y &=& g(x,u) \end{array}$$

with

$$egin{array}{lll} f(( heta,\omega),c) &=& \left(\omega,-(b/m\ell^2)\omega-(g/\ell)\sin heta+c/m\ell^2
ight) \ g(( heta,\omega),c) &=& heta. \end{array}$$

Let  $heta_e$  in  $\mathbb{R}$ . If  $c=c_e$ , the state  $x_e:=( heta_e,0)$  is an equilibrium if and only if  $f(( heta_e,0),c_e)=0$ , that is

$$egin{bmatrix} 0 \ 0 - (g/\ell)\sin heta_e + c_e/m\ell^2 \end{bmatrix} = egin{bmatrix} 0 \ 0 \end{bmatrix}$$

which holds if and only if

$$c_e = mg\ell\sin\theta_e$$
.

### 3. 🔓

We have

$$A = rac{\partial f}{\partial x}(x_e, c_e) = egin{bmatrix} 0 & 1 \ -(g/\ell)\cos heta_e & -(b/m\ell^2) \end{bmatrix}$$

$$B=rac{\partial f}{\partial u}(x_e,u_e)=egin{bmatrix} 0 \ 1/m\ell^2 \end{bmatrix}$$

$$C=rac{\partial g}{\partial x_e}(x_e,u_e)=egin{bmatrix}1\0\end{bmatrix},\ D=rac{\partial g}{\partial u_e}(x_e,u_e)=0$$

Thus,

$$egin{aligned} rac{d}{dt}\Delta heta &pprox \Delta\omega \ rac{d}{dt}\Delta\omega &pprox -(g/\ell)\cos( heta_e)\Delta heta - (b/m\ell^2)\Delta\omega + \Delta c/m\ell^2 \end{aligned}$$

and obviously, as far as the output goes,

$$\Delta \theta \approx \Delta \theta$$
.